
Theoretical Relationship Between Hot-Side Temperature (T_h) and Maximum Temperature Difference (ΔT_{\max})

A First-Principles Derivation for Thermoelectric Cooler Modules
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1. Introduction

In thermoelectric cooling (TEC) modules, the maximum achievable temperature difference ΔT_{\max} is not a fixed constant — it depends on the absolute hot-side temperature T_h at which the module operates. Engineers frequently observe that a TEC rated for $\Delta T_{\max} \approx 68^\circ\text{C}$ at $T_h = 27^\circ\text{C}$ will reach roughly 76°C at $T_h = 50^\circ\text{C}$ and over 96°C at $T_h = 100^\circ\text{C}$. The hardware has not changed; the physics simply favors warmer operation.

This Technical Note derives the relationship from first principles, then explains in plain language why higher T_h produces a larger ΔT_{\max} . A short cartoon and a single intuitive analogy capture the mechanism. Figure 1 plots the predicted curve, normalized to the datasheet reference ($T_h = 27^\circ\text{C}$), together with the optimum drive current I_{TEC} also normalized to its datasheet value I_{\max} .

2. The Thermoelectric Energy Balance Equation

The net cooling capacity Q_c at the cold junction is governed by three competing physical phenomena:

Peltier cooling — active heat pumping proportional to current I and cold-side absolute temperature T_c : $Q_{\text{peltier}} = \alpha \cdot T_c \cdot I$. This term is the root cause of the T_h -dependence.

Joule heating — parasitic resistive dissipation $I^2 R$ inside the pellets; one half flows back into the cold side.

Fourier conduction — parasitic thermal back-conduction $K \cdot \Delta T$ from hot to cold side through the pellets.

The cold-side energy balance is therefore:

$$Q_c = \alpha T_c I - \frac{1}{2} I^2 R - K \Delta T \quad (1)$$

where α is the Seebeck coefficient (V/K), T_c is the cold-side absolute temperature (K), I is the operating current (A), R is the electrical resistance (Ω), K is the thermal conductance (W/K), and $\Delta T = T_h - T_c$.

3. Deriving ΔT_{\max} — Optimum Drive Current

ΔT_{\max} occurs when $Q_c = 0$ — the operating point where all of the module's pumping capacity is consumed by its own internal losses. Setting $Q_c = 0$ in Equation (1):

$$0 = \alpha T_c I - \frac{1}{2} I^2 R - K \Delta T_{\max} \quad (2)$$

Solving for ΔT_{\max} as a function of I :

$$\Delta T_{\max} = \left(\alpha T_c I - \frac{1}{2} I^2 R \right) / K \quad (3)$$

Optimize the current by differentiating Equation (3) and setting $d(\Delta T_{\max})/dI = 0$:

$$d(\Delta T_{\max})/dI = (\alpha T_c - I R) / K = 0 \quad (4)$$

which immediately gives the optimum current:

$$I_{\text{opt}} = \alpha T_c / R \quad (5)$$

Note: I_{opt} scales linearly with T_c , so the optimum drive current itself rises with T_h . Figure 1 plots $I_{\text{TEC}} / I_{\text{max}}$ (right axis), normalized to the datasheet reference at $T_h = 27^\circ\text{C}$.

Substituting I_{opt} back into Equation (3) and simplifying:

$$\Delta T_{\max} = \frac{1}{2} \cdot (\alpha^2 / R K) \cdot T_c^2 \quad (6)$$

4. The Figure of Merit Z and the Cooling-Side Master Equation

The grouped material parameter $\alpha^2 / (R K)$ is the thermoelectric Figure of Merit Z , with units of K^{-1} :

$$Z = \alpha^2 / (R K) \quad [\text{K}^{-1}] \quad (7)$$

Equation (6) collapses to the master form for the cooling side:

$$\Delta T_{\max} = \frac{1}{2} \cdot Z \cdot T_c^2 \quad (8)$$

This is the central result of the derivation. It states that the maximum temperature difference scales linearly with Z and with the *square of the cold-side absolute temperature*. For Bi_2Te_3 at room temperature, $Z \approx 2.5 \times 10^{-3} \text{K}^{-1}$.

5. Why Higher T_h Helps — the Physics in One Sentence

Equation (8) is in terms of T_c , but engineers control T_h . Because $\Delta T = T_h - T_c$, raising T_h raises T_c as well — even at the maximum- ΔT operating point. And the master equation says cooling power scales with the *square of T_c* . So:

higher T_h → higher T_c → more Peltier heat per amp → larger ΔT_{\max}

6. An Intuitive Analogy — Ants Carrying Heat Buckets

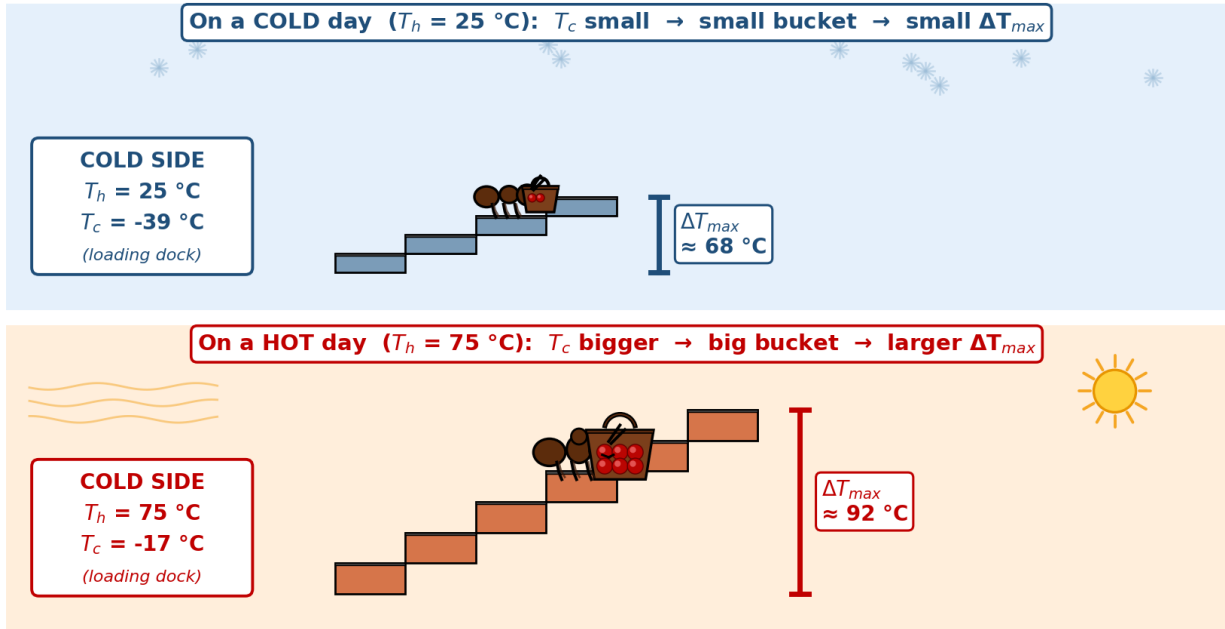
Picture each electron as a tiny ant carrying a bucket of heat from the cold side up a flight of stairs to the hot side. The bucket capacity is set by physics: each ant can carry an amount of heat proportional to the absolute temperature where it picks up the load — $qe = \alpha \cdot T_c$ per coulomb (entropy times temperature, the Peltier cooling rate per amp).

On a cold day (low T_h , and so a low T_c), each ant's bucket is small. The ant struggles up the stairs and barely moves any heat per trip; the module's pumping margin is modest, and ΔT_{\max} stays around 68°C .

On a hot day (high T_h , and so a high T_c), each ant carries a much larger bucket. Same number of ants (same drive current I), but more heat per trip overcomes more parasitic loss before the cooling capacity is exhausted. ΔT_{\max} climbs to roughly 92°C — even though the hardware is identical.

That is the entire mechanism behind $\Delta T_{max} = \frac{1}{2} Z T_c^2$. The cooling-side master equation says cooling power scales with the square of T_c , and T_c rises with T_h , so warmer ambient operation lets each electron move more heat per trip. Figure 2 shows the picture.

Each electron is an ant; bucket per ant $q_e = \alpha T_c$, so it grows with T_c



Master equation: $\Delta T_{max} = \frac{1}{2} Z T_c^2$ — bigger $T_c \rightarrow$ bigger bucket \rightarrow larger achievable temperature span

Figure 2. Ants carrying heat buckets. On a cold day each ant has a small bucket and barely moves any heat ($\Delta T_{max} \approx 68\text{ }^\circ\text{C}$). On a hot day each ant is loaded up with a much larger bucket and the module reaches $\Delta T_{max} \approx 92\text{ }^\circ\text{C}$ — same hardware, bigger cargo per trip.

7. Expressing ΔT_{max} in Terms of T_h

Equation (8) is in terms of T_c , but for engineering use we need it in terms of T_h (which the heat sink fixes). Substituting $T_c = T_h - \Delta T_{max}$:

$$\Delta T_{max} = \frac{1}{2} Z (T_h - \Delta T_{max})^2 \quad (9)$$

Expanding and rearranging into a standard quadratic in ΔT_{max} :

$$\frac{1}{2} Z \Delta T_{max}^2 - (Z T_h + 1) \Delta T_{max} + \frac{1}{2} Z T_h^2 = 0 \quad (10)$$

The physically meaningful root (the one that keeps $T_c > 0\text{ K}$) is:

$$\Delta T_{max} = T_h + 1/Z - \sqrt{(2T_h/Z + 1/Z^2)} \quad (11)$$

A simplified approximation, valid only when $Z T_h \ll 1$, is sometimes seen:

$$\Delta T_{max} \approx \frac{1}{2} Z T_h^2 \quad (12)$$

Use Equation (11) for any quantitative work. For real Bi_2Te_3 the product ZTh is of order 1 (not $\ll 1$) at typical operating temperatures, so Equation (12) overestimates the exact value substantially. It is shown here for completeness only and is not plotted in Figure 1.

8. Numerical Example and Graphical Comparison

Using $Z = 2.5 \times 10^{-3} \text{ K}^{-1}$ (typical for Bi_2Te_3), Equation (11) gives the values in Table 1. The *normalized* columns express $\Delta T_{\max}(Th)$ as a multiple of ΔT_{\max} at $Th = 27^\circ\text{C}$ and $\text{ITEC}(Th)$ as a multiple of I_{\max} at $Th = 27^\circ\text{C}$ — multiply each ratio by the corresponding datasheet value to predict performance at any Th .

Table 1. ΔT_{\max} and ITEC predictions for Bi_2Te_3 ($Z = 2.5 \times 10^{-3} \text{ K}^{-1}$).

Th ($^\circ\text{C}$)	Th (K)	T_c (K)	ΔT_{\max} ($^\circ\text{C}$)	$\Delta T_{\max} / \Delta T_{\max,ref}$	ITEC (A)	ITEC / I_{\max}	Notes
-20	253.15	202.10	51.05	0.755	5.56	0.869	
0	273.15	215.24	57.91	0.857	5.92	0.926	
25	298.15	231.28	66.87	0.989	6.37	0.995	
27	300.15	232.55	67.60	1.000	6.40	1.000	ref
50	323.15	246.93	76.22	1.128	6.80	1.062	
75	348.15	262.21	85.94	1.271	7.22	1.128	
100	373.15	277.14	96.01	1.420	7.63	1.192	
120	393.15	288.85	104.30	1.543	7.95	1.242	

Reference row at $Th = 27^\circ\text{C}$ anchors both normalized columns. Multiply each ratio by your module's datasheet ΔT_{\max} (or I_{\max}) to predict performance at the actual operating Th .

Figure 1 plots the same data over a continuous Th sweep from -20°C to 120°C . The blue curve is the normalized $\Delta T_{\max}(Th)$ ratio (left axis). The brown dot-dashed curve is ITEC / I_{\max} (right axis), also normalized to the $Th = 27^\circ\text{C}$ reference. The red dots mark the datasheet anchor where both curves equal 1.000 by definition.

Figure 1. Normalized ΔT_{max} and I_{TEC}/I_{max} vs. T_h for Bi_2Te_3 ($Z = 2.5 \times 10^{-3} K^{-1}$, reference: $T_h = 27^\circ C$)

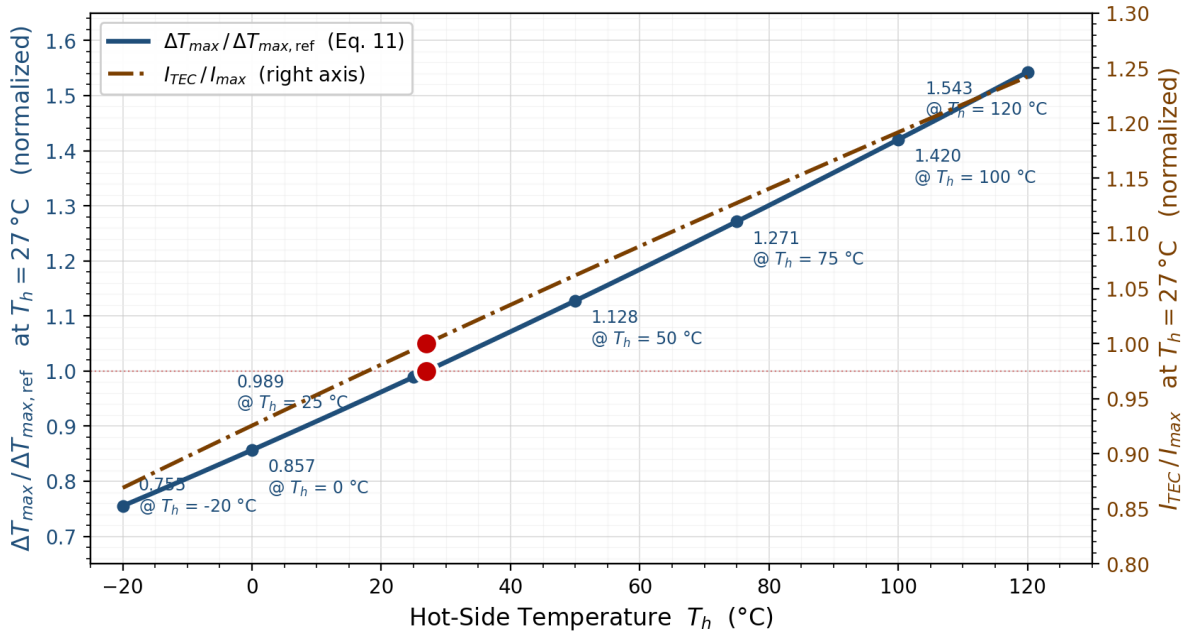


Figure 1. Normalized ΔT_{max} (left axis) and I_{TEC}/I_{max} (right axis) vs. T_h for Bi_2Te_3 , $Z = 2.5 \times 10^{-3} K^{-1}$, reference $T_h = 27^\circ C$.

9. The TEC Drive Current at ΔT_{max}

Equation (5) gives $I_{opt} = \alpha T_c / R$, and T_c rises with T_h . In Figure 1 the brown dot-dashed curve shows I_{TEC}/I_{max} swinging from about 0.87 at $T_h = -20^\circ C$ up to 1.24 at $T_h = 120^\circ C$ — a 40 % swing. Two practical implications follow:

- 1. Drive-current sizing.** A controller that current-limits to the datasheet I_{max} value at $27^\circ C$ will under-drive the module at $T_h = 100^\circ C$ and miss the larger ΔT_{max} that the physics would otherwise allow. ATI controllers (TEC18V15A, TEC50V20ACH) support adjustable current limits via the IMS/VLM pins so the user can match I_{opt} to the actual operating T_h .
- 2. Power-supply sizing.** The TEC voltage at I_{opt} is roughly $V_{opt} \approx \alpha T_c + I_{opt} R \approx 2 \alpha T_c$, so peak input power demand at high T_h is materially larger than at room temperature. Always size VPS for the worst-case (highest- T_h) operating point.

10. Practical Implications for System Designers

Specify ΔT_{max} at the actual operating T_h , not the datasheet reference. Datasheet ΔT_{max} values are quoted at $T_h = 27^\circ C$ by convention, but most real systems run hotter. Read the normalization ratio from Figure 1 (or Table 1), multiply by your datasheet value, and use that for design.

Use Equation (11), not Equation (12). The simplified parabolic form overshoots the exact value substantially for any modern TE material at typical T_h , so it is omitted from Figure 1 and should be reserved for back-of-envelope estimates only.

Match I_{opt} to T_h . If the system runs over a wide ambient range, set the controller current limit to track $I_{opt}(T_h)$ using a T_h sense input. This squeezes out the last 5 - 15 % of ΔT_{max} that a fixed current limit leaves on the table.

Keep T_c above the dew point if condensation is a concern. In humid environments, the cold side at ΔT_{\max} can drop below the dew point and accumulate water. Run the module sub- ΔT_{\max} , or seal/desiccate the cold-side enclosure.

11. Summary

The maximum temperature difference of a thermoelectric cooler is governed by the cooling-side master equation $\Delta T_{\max} = \frac{1}{2} Z T_c^2$. Because T_c rises with T_h at the ΔT_{\max} operating point, the achievable temperature span itself rises with T_h . Each electron carries Peltier heat αT_c per coulomb, so the 'bucket per ant' grows with absolute temperature; that is the whole story behind the T_h -dependence.

Equation (11) gives the exact closed-form $\Delta T_{\max}(T_h)$; use it. At the optimum operating point the drive current $I_{opt} = \alpha T_c / R$ also rises with T_h — size your controller and supply accordingly.

References

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